

***EE392 - ELECTRICAL ENGINEERING  
PRACTICE***

***MECHANICAL COMPONENT***

**Static Mechanics:**

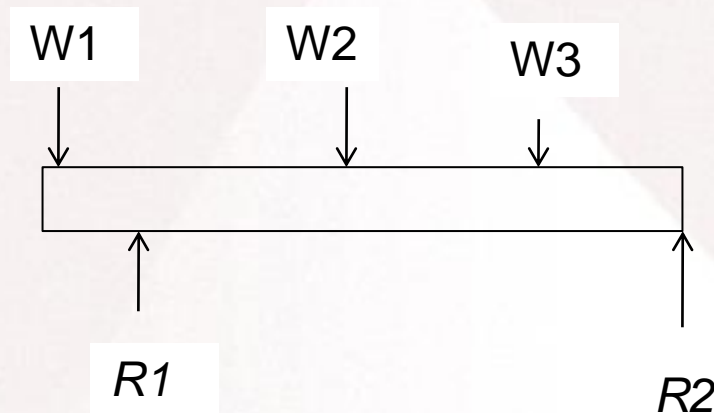
**Shearing Force and Bending Moments in Beams**

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## Shearing Force in beams

### 4.1 : Shearing Force

- The shearing force at any section of a beam represents the tendency for the portion of the beam to one side of the section to slide or shear laterally relative to the other portion.



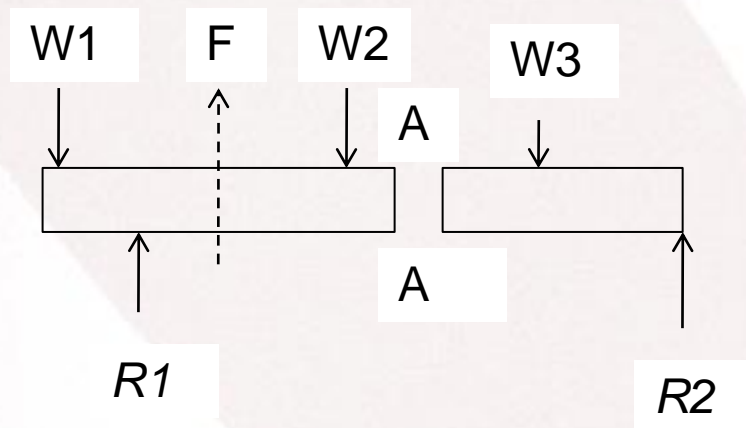
A beam carrying loads  $W_1$ ,  $W_2$ , and  $W_3$  is simply supported at two points.

The reactions at the supports being  $R_1$  and  $R_2$ .

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## Shearing force continued...

- Now imagine the beam to be divided into two portions by a section at AA.

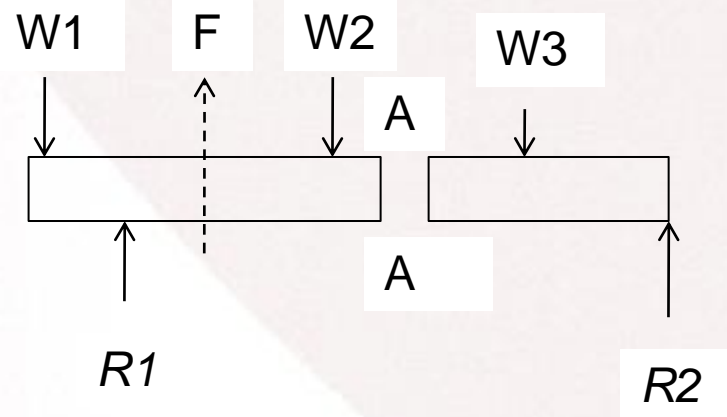


- The resultant of the loads and reactions to the left of AA is  $F$  vertically upwards, and since the whole beam is in equilibrium, the resultant of the forces to the right of AA must also be  $F$ , acting downwards.

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## Shearing force continued..

- $F$  is called the Shearing Force (abbrev. S.F.) at the section AA and maybe defined as follows: **the shearing force at any section of a beam is the algebraic sum of the lateral components of the forces acting on either side of the section.**



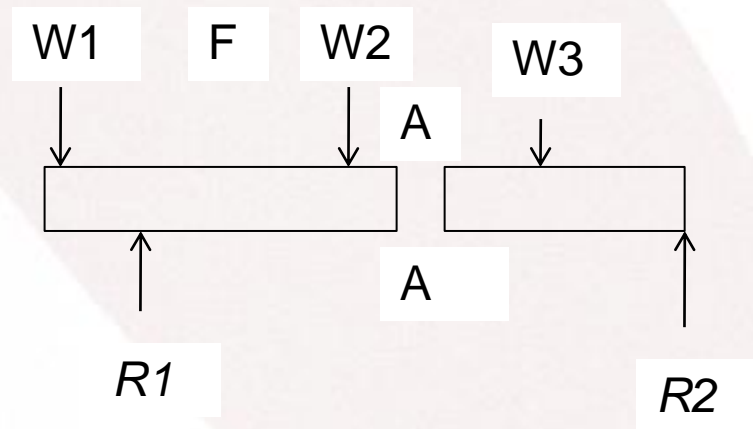
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- **Shearing force continued ...**
- Where a force is in neither the axial nor the lateral direction it must be resolved in the usual way, the lateral component being taken into account in the shearing.
- Shearing force will be considered positive when the resultant of the forces to the left is upwards, or to the right is downwards.
- **Shearing force diagram** is one which shows the variation of shearing force along the length of the beam.

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## 4.2 Bending Moment

- In a similar manner it can be argued that the moment about the section AA of the forces to the left is  $M$  clockwise, then the moment of the forces to the right of AA must be  $M$  anticlockwise.



- $M$  is called the **Bending Moment** (abbr. B.M.) at AA, and is defined as: the algebraic sum of the moments about the section of all the forces acting on either side of the section.

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- **Bending Moment continued ...**
- Bending moment will be considered positive when the moment on the left portion is clockwise, and on the right portion anticlockwise.
- This is referred to as sagging bending moment since it tends to make the beam concave upwards at AA.
- Negative bending moment is termed hogging.
- A **bending moment diagram** is one which shows the variation of bending moment along the length of the beam.

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## **4.3 Types of Load**

- A **beam** is normally horizontal, the loads being vertical, other cases which occur being looked upon as exceptions
- A **concentrated load** is one which is considered to act at a point, although in practice it must really be distributed over a small area.
- A **distributed load** is one which is spread in some manner over the length of the beam.
- The **rate of loading  $w$**  maybe uniform, or may vary from point to point along the beam.

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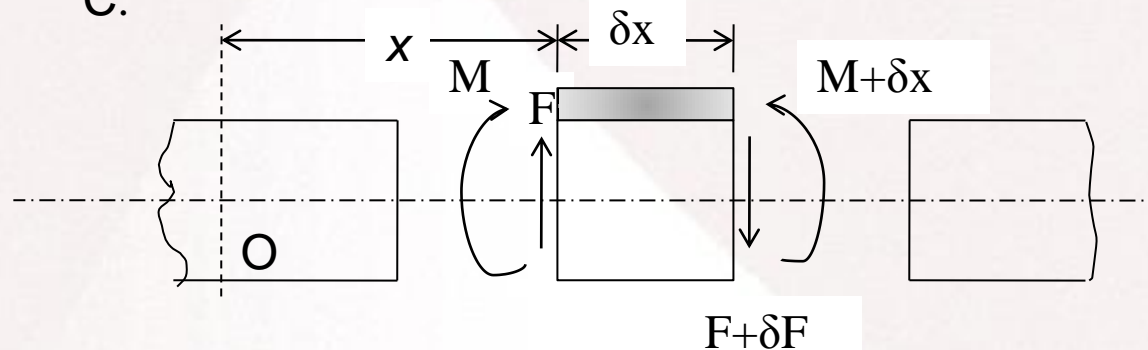
## **4.4 Types of Support**

- A **simple or free support** is one on which the beam is rested, and which exerts a reaction on the beam.
- A **built-in or encastre support** is frequently met with, **the effect being to fix the direction of the beam at the support.**
- In order to do this the support must exert a ‘fixing’ moment  $M$  and a reaction  $R$  on the beam.
- A beam thus fixed at one end is called a cantilever

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## 4.5 Relations between $w$ , $F$ , and $M$

- Consider a short length  $\delta x$  imagined to be a slice cut out from a loaded beam at a distance  $x$  from a fixed origin  $O$
- Let the shearing force at the section  $x$  be  $F$ , and at  $x + \delta x$  be  $F + \delta F$ .
- Similarly the bending moment is  $M$  at  $x$ , and  $M + \delta M$  at  $x + \delta x$ .
- If  $w$  is the mean rate of loading on the length  $\delta x$ , the total load is  $w\delta x$ , acting approximately (exactly, if uniformly distributed) through the center  $C$ .



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## **Relations between $w$ , $F$ , and $M$ Continued ...**

- The elements must be in equilibrium under the action of these forces and couples, and the following equations are obtained.
- Taking moments about C:
- $M + F \cdot \delta x / 2 + (F + \delta F) \delta x / 2 = M + \delta M$
- Neglecting the product  $\delta F \cdot \delta x$ , and taking the limit, gives
- $F = dM/dx$  -----(1)

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- **Relations between  $w$ ,  $F$ , and  $M$  Continued ...**
- Resolving vertically

$$w\delta x + F + \delta F = F$$

$$w = - dF/dx \text{ ----- (2)}$$

$$= - d^2M/dx^2 \text{ from (1) ----- (3)}$$

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- **Relations between  $w$ ,  $F$ , and  $M$  Continued ...**
- From equation (1) it can be seen that, if  $M$  is varying continuously, zero shearing force corresponds to maximum or minimum bending moment.
- $F = dM/dx$  -----(1)
- It will be seen later however that “peaks” in the bending moment diagram frequently occur at concentrated loads or reactions, and are not then given by  $F = dM/dx = 0$ , although they may represent the greatest bending moment on the beam.
- Consequently it is not always sufficient to instigate the points of zero shearing force when determine the maximum bending moment.
- At a point on the beam where the type of bending is changing from sagging to hogging, the bending moment must be zero, and this is called a point of inflection or contraflexure.

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- **Relations between  $w$ ,  $F$ , and  $M$  Continued ...**
- By integrating equation (1) between two values of  $x = a$  and  $x = b$ , then  $M_b - M_a = \int F dx$
- Showing that the increase in bending moment between two sections is given by the area under the shearing force diagram

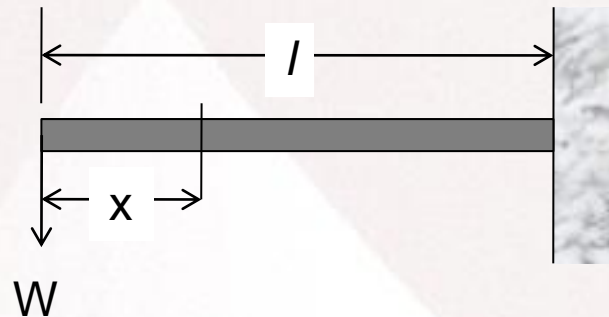
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- **Relations between w, F, and M Continued ...**
- Similarly, integrating equation (2)
- $F_a - F_b = \int w dx$
- = the area under the load distribution diagram.
- Integrating equation (3) gives
- $M_a - M_b = \iint w dx \cdot dx$
- These relations prove very valuable when the rate of loading cannot be expressed in algebraic form, and provide a means of graphical solution.

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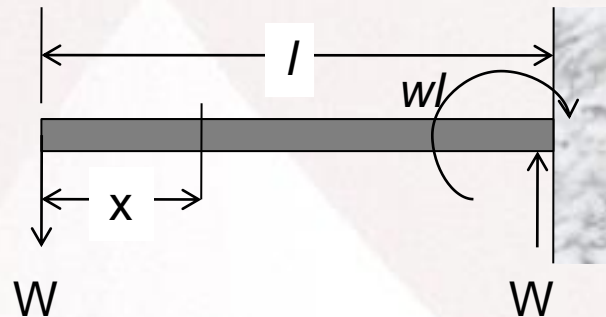
## Examples

- 4.6 Concentrated Loads
- **Example 1.** A cantilever of length  $l$  carries a concentrated load  $W$  at its free end. Draw the S.F. and B.M. diagrams



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- **Example 1 continued ...**
- **Solution:**
- At a section a distance  $x$  from the free end, consider the forces to the left.
- Then  $F = -W$  and is constant along the whole beam (i.e. for all  $x$ ).

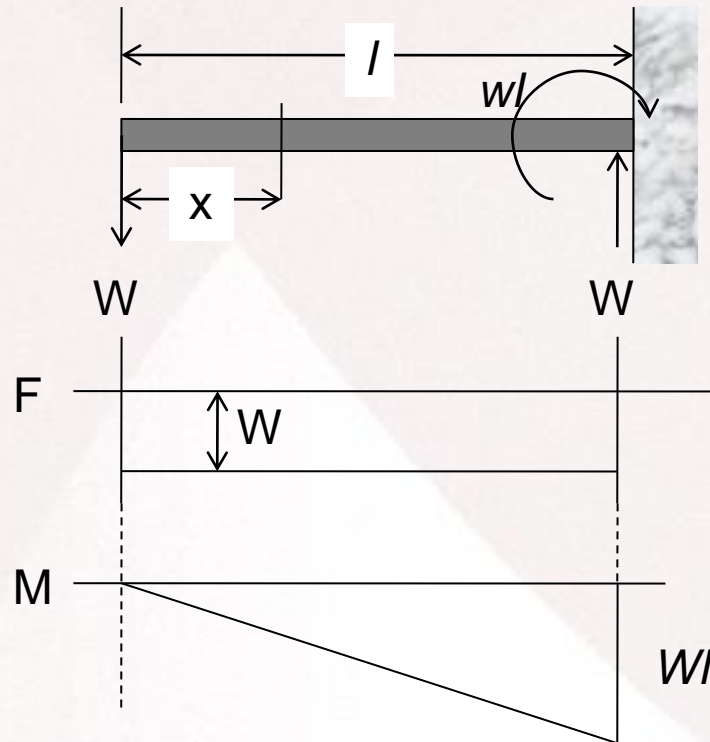


- Taking moments about the section gives  $M = -Wx$ , so that the maximum bending moment occurs at the fixed end., i.e.  $M_{\max} = Wl$  (hogging)
- From equilibrium considerations, the fixing moment applied at the built in end is  $Wl$ , and the reaction is  $W$ .

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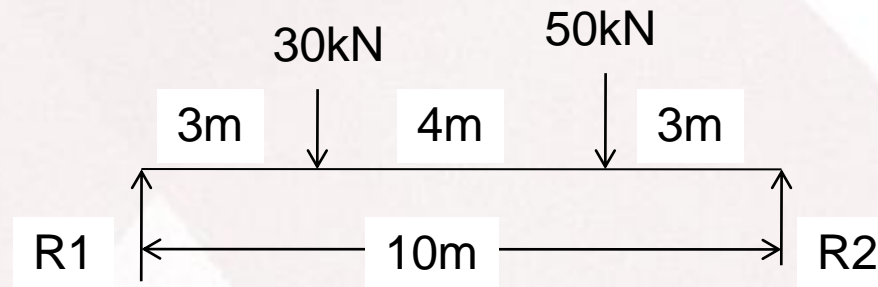
## Concentrated load continued

- The S.F. and B.M. diagrams are therefore as shown:



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- **Example 2.** A beam 10m long is simply supported at its ends and carries concentrated loads of 30kN and 50kN at distances of 3 m from each end. Draw the S.F. and B.M. diagrams.



- **SOLUTION**
- First calculate the reactions R1 and R2 at the supports

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- By moments about R2:

- $R1 \times 10 = 30 \times 7 + 50 \times 3$

- Therefore  $R1 = 36\text{kN}$

- and  $R2 = 30 + 50 - R1 = 44\text{kN}$

- Let  $x$  be the distance of the section from the left – hand end

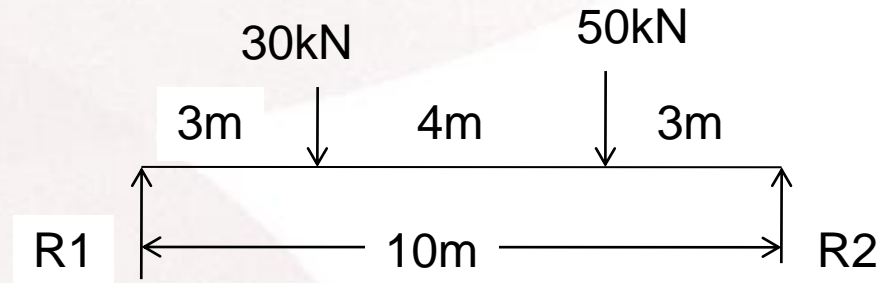
- **Shearing force:**

$$0 < x < 3, \mathbf{F} = \mathbf{R1} = 36\text{kN}$$

$$3 < x < 7, \mathbf{F} = \mathbf{R1} - 30 = 6\text{kN}$$

$$7 < x < 10, \mathbf{F} = \mathbf{R1} - 30 - 50 = -44\text{kN}$$

- Note that the last value =  $-R2$ , which provides a check on the working



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- **Bending Moment:**

$$0 < x < 3, M = R1x = 36x \text{ kNm}$$

$$3 < x < 7, M = R1x - 30(x-3) = 6x + 90 \text{ kNm}$$

$$7 < x < 10, M = R1x - 30(x-3) - 50(x-7) \\ = -44x + 440 \text{ kNm}$$