

A large curved dam with water behind it, surrounded by green hills. The dam is a prominent feature in the center of the image, with water visible behind it. The surrounding landscape is lush with green trees and vegetation. The text is overlaid on the image in a yellow, serif font.

Fluid Mechanics CEE 3311

LECTURE 4

Forces on curved surfaces

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Forces on curved surfaces

- We do not use a direct method of integration to find the force due to the hydrostatic pressure on a curved surface.
- Rather, a *free body diagram* that contains the curved surface and the liquids directly above or below the curved surface is identified.
- Such a free-body diagram contains *only plane surfaces* upon which unknown fluid forces act; these unknown forces can be found as in the preceding lecture.

Example

- Let us determine the force (P) of the curved gate on the stop.

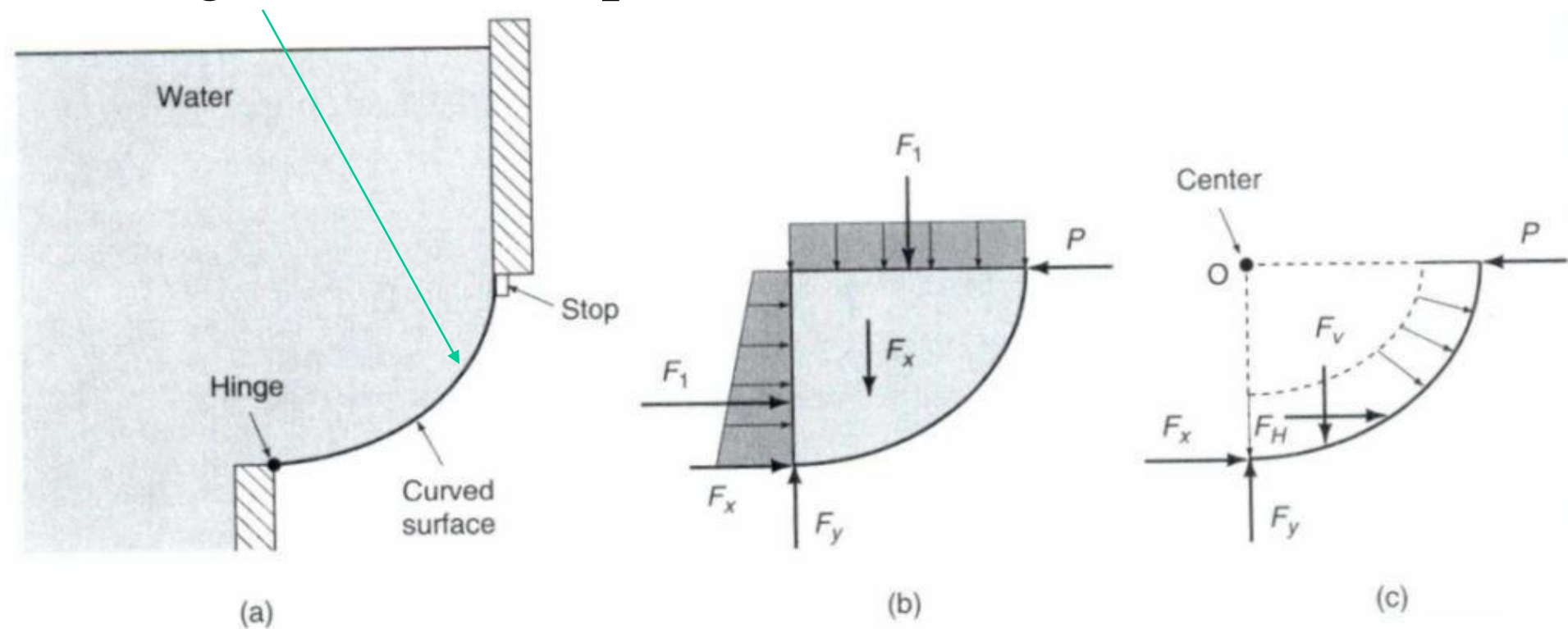


Fig 4.1 Forces on a curved surface: a) curved surface b) free-body diagram of water and gate c) free-body diagram of gate only

- The free-body diagram includes the gate and some of the water contained directly above the gate.
- The forces F_x and F_y are the horizontal and vertical components, respectively of the force acting on the hinge.
- The forces F_1 and F_2 are due to the surrounding water and are the resultant forces of the pressure distributions shown.
- The body force F_w is due to the weight of the water shown.
- By summing moments about an axis through the hinge, we can determine the force P acting on the stop.

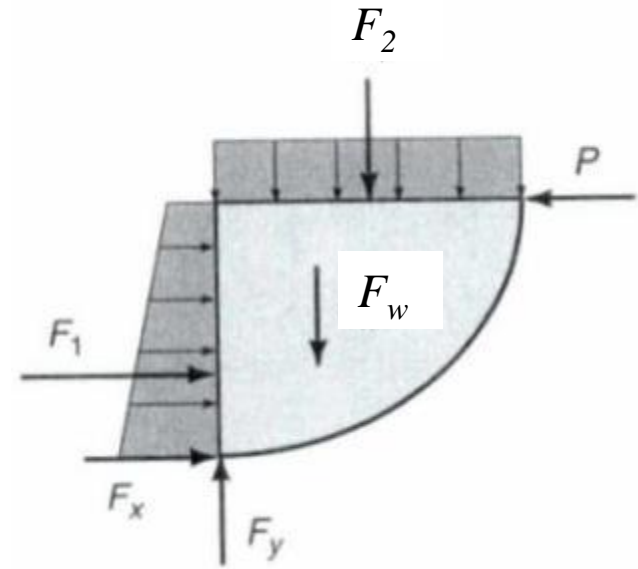


Fig 4.2 Free-body diagram of water and gate

- Consider a free-body diagram of the gate only.
- The horizontal force F_H acting on the gate in Fig 4.3 is equal to F_1 of Fig 4.2.
- The component F_V in fig 4.3 is equal to the combined force $F_2 + F_W$ of Fig 4.2
- Now, $F_H + F_V$ are due to the differential pressure forces acting on the circular arc; each differential pressure force acts through the center of the circular arc, O (since forces act normal to surface).
- Hence, the resultant force $F_H + F_V$ (this the vector addition) must act through the center.
- Consequently, we can locate the components $F_H + F_V$ at the center of the quarter circle, resulting in a much simpler problem.

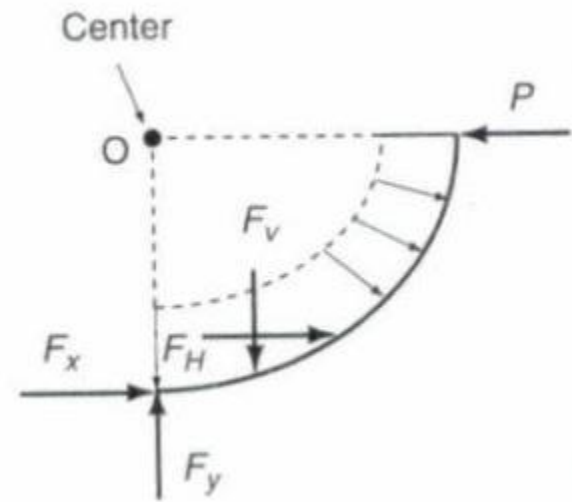


Fig 4.3 Free-body diagram of gate only

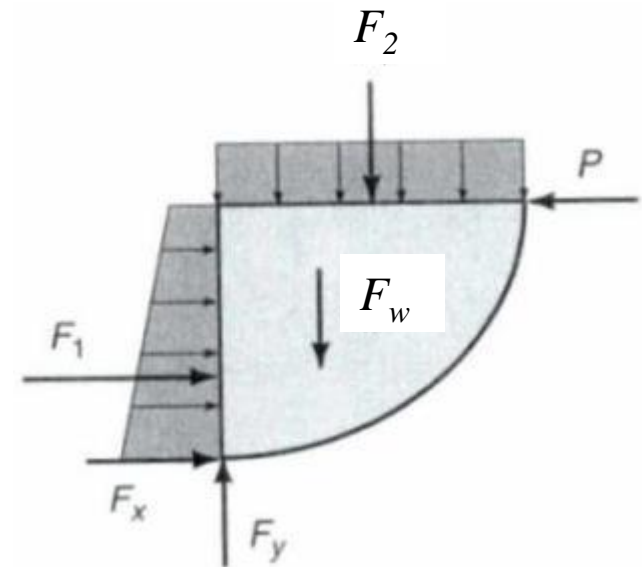


Fig 4.2 Free-body diagram of water and gate

Example 1

Calculate the force P necessary to hold the 4m-wide gate in the position shown in Fig 4.4 (a). Neglect the weight of the gate.

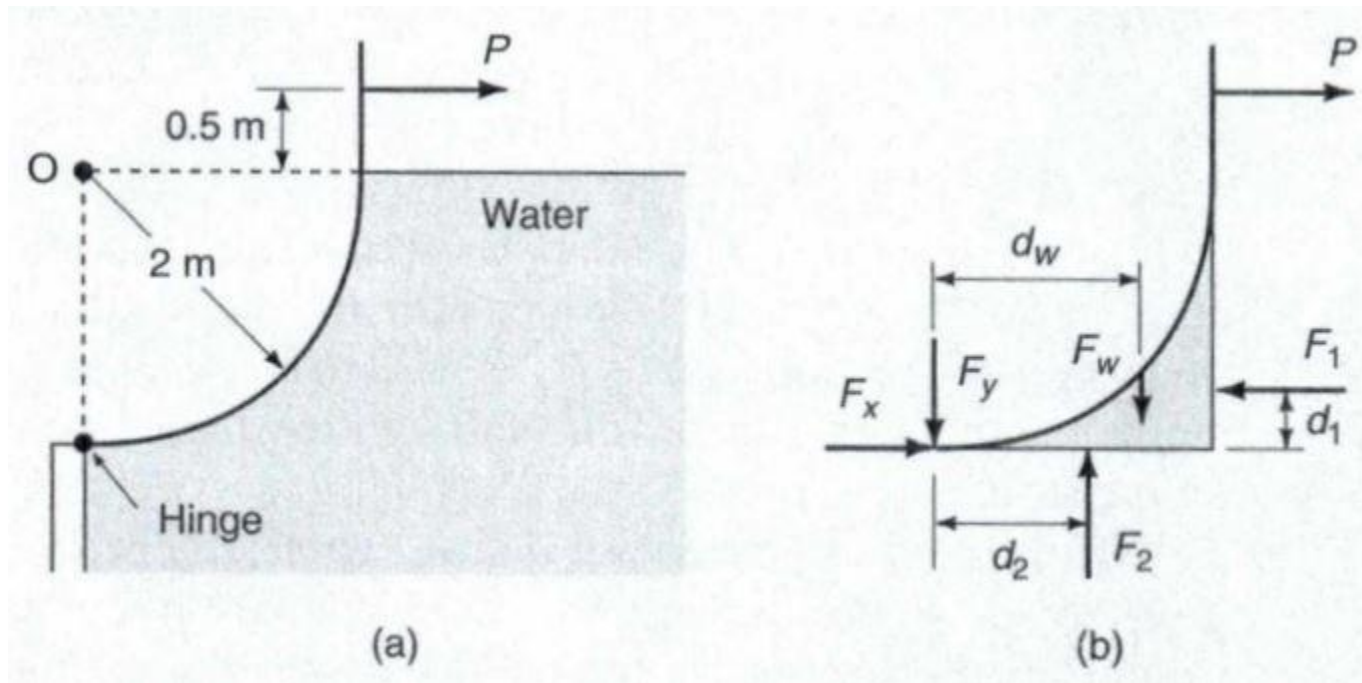


Fig 4.4 Example

Solution

The first step is to draw a free-body diagram of the gate and the water directly below the gate as shown in Fig 4.4 (b).

To calculate P , we must determine F_1, F_2, F_W, d_1, d_2 and d_W ; then moments about the hinge will allow us to find P . The force components are given by

$$F_1 = \bar{\gamma} h A$$

$$= 9810 \times 1 \times 8 = 78\,480 \text{ N}$$

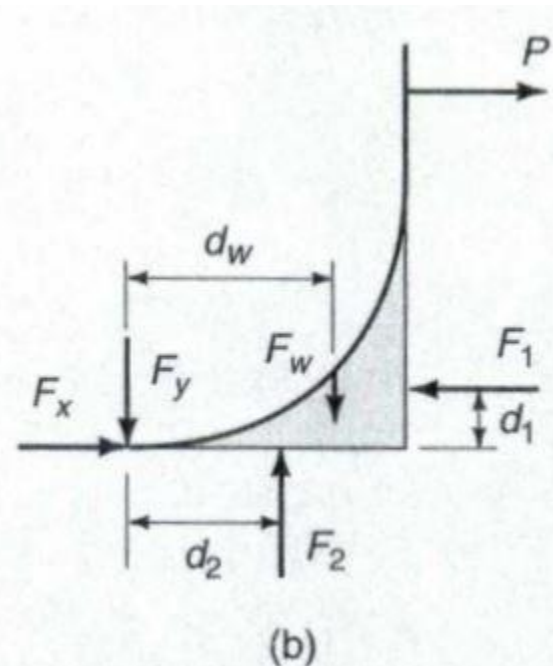
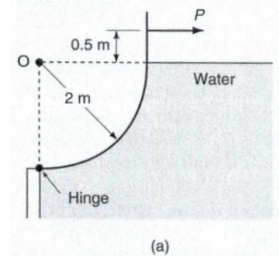
$$F_2 = \bar{\gamma} h A$$

$$= 9810 \times 2 \times 8 = 156\,960 \text{ N}$$

$$F_W = \gamma V$$

$$= 9810 \times 4(4 - \pi) = 33\,700 \text{ N}$$

You can also
use $p = \gamma h$



The distance d_w is the distance to the centroid of the volume. It can be determined by considering the area as the difference of a square and a quarter circle as shown in Fig 4.4 c-e.

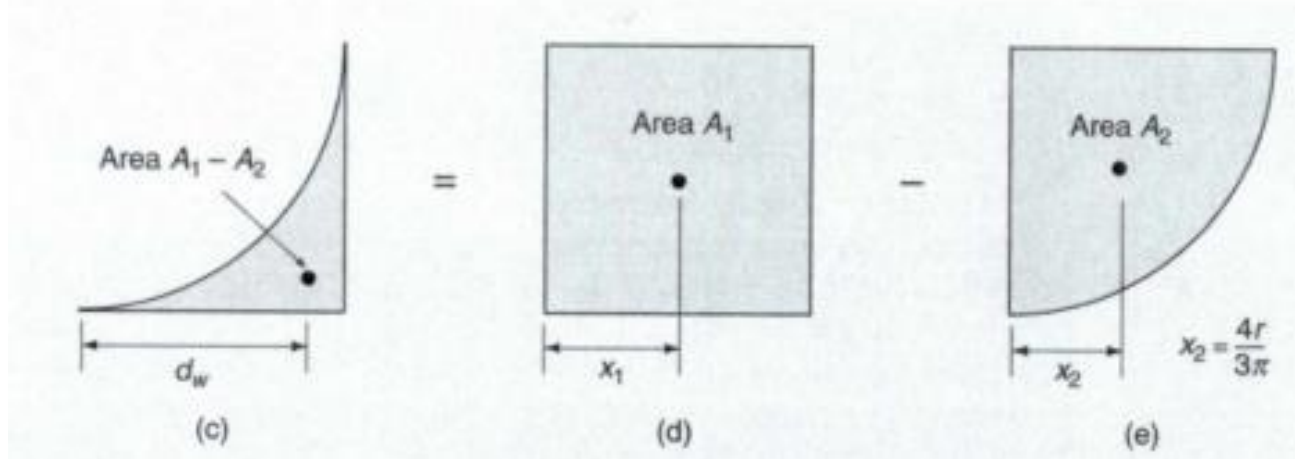


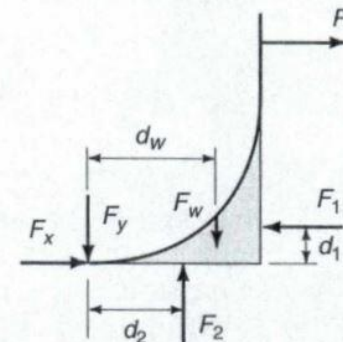
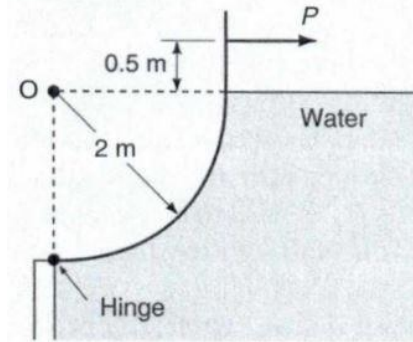
Fig 4.4 Example

Moments of area yield

$$\begin{aligned}
 d_w(A_1 - A_2) &= x_1A_1 - x_2A_2 \\
 d_w &= \frac{x_1A_1 - x_2A_2}{A_1 - A_2} \\
 &= \frac{1 \times 4 - (4 \times \frac{2}{3}\pi) \times \pi}{4 - \pi} = 1.553 \text{ m}
 \end{aligned}$$

The distance $d_2 = 1\text{ m}$ and because F_1 is due to a triangular pressure distribution (see Fig 4.4 (d)), d_1 is given by

$$d_1 = \frac{1}{3}(2) = 0.667 \text{ m}$$

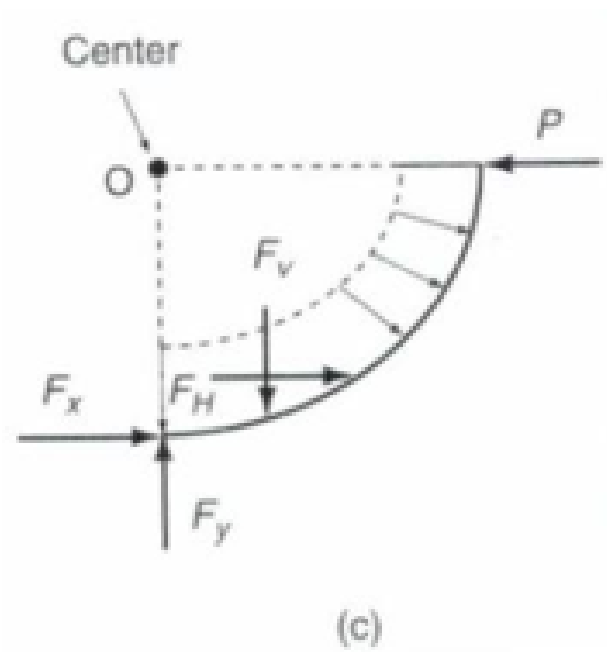


Summing moments about the frictionless hinge^(a) gives

$$2.5P = d_1F_1 + d_2F_2 - d_wF_w$$

$$P = \frac{0.667 \times 78.5 + 1 \times 157.0 - 1.553 \times 33.7}{2.5} = 62.8 \text{ kN}$$

- Rather than the somewhat tedious procedure above, we could observe that all the infinitesimal forces that make up the resultant force ($F_H + F_V$) acting on the circular arc pass through the center O, as noted in Fig 4.3 (c).

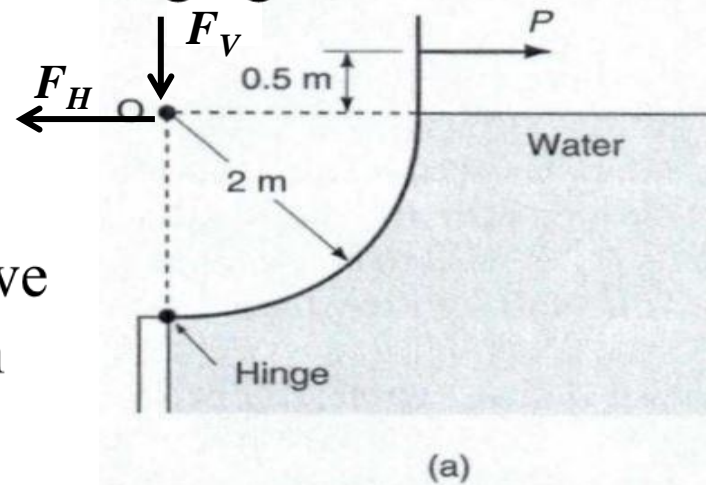


- Since each infinitesimal force must also pass through the center, the resultant force *must also* pass through the center.
- Hence, we could have located the resultant force ($F_H + F_V$) at point O.
- If $F_H + F_V$ were located at O, F_V would pass through the hinge, producing no moment about the hinge. Then realising that $F_H = F_1$ and summing moments about the hinge gives

$$2.5P = 2F_H$$

$$P = 2 \times \frac{78.48}{2.5} = 62.8 \text{ kN}$$

- Therefore
- This was obviously much simpler. All we needed to do was calculate F_H and then sum moments!



Example 2: Arch dam

Kariba Dam



[Kariba Dam](#) on the Zambezi River.

Credit: Ben Bird

Normal plane area where hydrostatic force is acting is

$$2r \sin \frac{\theta}{2}$$

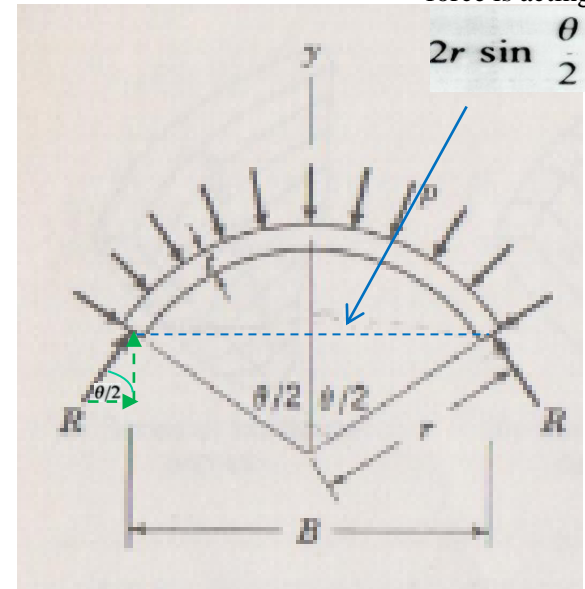


Figure 8-11 Free-body diagram of an arch rib.

$$H_n = \gamma h 2r \sin \frac{\theta}{2}$$